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# ***U.S. PATENT APPLICATION***

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***Invention:*** FLUID INJECTION NOZZLE

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## ***SPECIFICATION***

## FLUID INJECTION NOZZLE

### CROSS REFERENCE TO RELATED APPLICATION

This application is based upon, claims the benefit of  
5 priority of, and incorporates by reference, the contents of  
Japanese Patent Application No. Hei 11-224141 filed on August 6,  
1999, and is a divisional application of nonprovisional application  
number 10/141,553 filed on May 9, 2002.

### BACKGROUND OF THE INVENTION

#### 1. Field of the Invention:

The present invention relates to a fluid injection nozzle having  
a plate in which a fluid injection hole is formed. For instance, the  
present invention applies to a fuel injection valve for supplying fuel to  
15 an internal combustion engine (engine).

#### 2. Description of Related Art:

DE 19636396A1 discloses fuel injector having a plate in which a  
plurality of through holes are formed as fuel injection orifices. Such a  
plate type injectors are effective to generate a plurality of fuel jets.

20 In this arrangement, fuel flows along an inclined surface formed by a  
valve seat. However, some of the through holes are opened on an  
imaginary line where a surface of the plate crosses an extended line of  
the inclined surface. Therefore, fuel flowing along the inclined surface  
directly flows into the through holes. Therefore, fuel is insufficiently  
25 atomized.

USP 4,907,748, USP 5,762,272 and WO 98/34026 disclose the fuel  
injectors having flat chambers just upstream the through holes. Such a

chamber provides a compound fuel flow just upstream the through hole and is effective to atomize fuel. However, there is a possibility to spoil an atomization by a collision of injected fuel columns at just after the through holes. Here, the fuel column is a shape of fuel before fuel is atomized by collision with air. Further, a shape of a wall defining the chamber is important to define a fuel flow at an inlet of the through hole, since the fuel atomization is affected by the fuel flow flowing along the plate. However, WO 98/34026 does not provide a surface having a sufficient flatness and a size to atomize fuel.

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#### SUMMARY OF THE INVENTION

The present invention addresses these drawbacks by providing an improved fluid injection nozzle arrangement.

It is therefore an object of this invention to improve an atomization of fluid.

It is a further object of this invention to provide a fluid injection nozzle in which a collision of injected fluid columns is avoided.

According to a first aspect of the present invention, the fluid injection nozzle has a chamber for controlling a fluid flow to a through hole formed on a plate. Fluid flowing along an inner surface of a valve body is inclined to meet and collide at a center region of the plate. Therefore, fluid turns its direction and flows along the plate. Specifically, the chamber is flat and is extended more than a diameter of the through hole at an outside of the through hole. Therefore, fluid flows along the chamber for a sufficient distance and reaches the through hole from all directions and collides at an inlet of the through hole.

As a result, fluid injected from the through hole has a lot of  
turbulences and is finely atomized. Further, an inlet of the through  
hole opens at an outer area of a projected area which is defined by  
projecting a downstream end opening of the inner surface of the valve  
5 body. Therefore, the through holes are separately arranged to avoid a  
collision of columns of fluid injected from the through holes.

According to another aspect of the present invention, a plate  
has an inner through hole and an outer through hole located both side of  
an imaginary line. Here, the imaginary line is defined by crossing a  
10 surface of the plate and a line extended along the inner surface of the  
valve body. Therefore, the inner through hole and the outer through hole  
are mainly influenced by fluid flows having different directions. As a  
result, columns of injected fluid are directed in different directions  
and a collision of the columns is avoided.

#### BRIEF DESCRIPTION OF THE DRAWINGS

Other features and advantages of the present invention will be  
appreciated, as well as methods of operation and the function of the  
related parts, from a study of the following detailed description, the  
20 appended claims, and the drawings, all of which form a part of this  
application. In the drawings:

FIG. 1 is a partial sectional view of a nozzle portion of a fuel  
injector according to a first embodiment of the present invention;

FIG. 2 is a bottom view of a plate according to the first  
25 embodiment of the present invention;

FIG. 3 is a sectional view of the fuel injector according to the  
first embodiment of the present invention;

FIG. 4 is a partial sectional view of a nozzle portion of a fuel injector according to a second embodiment of the present invention;

FIG. 5 is a bottom view of a plate according to the second embodiment of the present invention;

5 FIG. 6 is a partial sectional view of a nozzle portion of a fuel injector according to a third embodiment of the present invention;

FIG. 7 is a partial sectional view of a nozzle portion of a fuel injector according to a fourth embodiment of the present invention;

10 FIG. 8 is a bottom view of a plate according to the fourth embodiment of the present invention;

FIG. 9 is a bottom view of a plate according to a fifth embodiment of the present invention;

FIG. 10 is a bottom view of a plate according to a sixth embodiment of the present invention; and

15 FIG. 11 is a bottom view of a plate according to a seventh embodiment of the present invention.

#### DETAILED DESCRIPTION OF PREFERRED EMBODIMENTS

20 Preferred embodiments of the present invention will be explained with reference to the drawings.

FIG. 1 through FIG. 3 shows a first embodiment of the present invention. In this embodiment, the present invention applies to a fuel injector for supplying fuel to an internal combustion engine such as a gasoline engine.

25 Referring to FIG. 3, the fuel injector 1 has a cylindrical stator core 30 for providing a fuel passage therein. The stator core 30 is connected to a first pipe 32 made of nonmagnetic material by a laser

welding. The first pipe 32 is connected to a second pipe 12 made of magnetic material by a laser welding. The second pipe 12 is connected to a valve body 13 by a laser welding. An electromagnetic coil having a spool 40 and a coil 41 is disposed on an outside of the stator core 30, and the first and second pipes 32 and 12. The coil 41 has a pair of terminals that are connected to connector pins 42 respectively. The coil 41 and the stator core 30 are covered with a resin 11 forming an outer body and a connector housing.

A movable valve member is disposed between the stator core 30 and the valve body 13. The movable valve member has a needle 20 and an armature core 31 made of a magnetic material. The armature core 31 is connected to an upper end of the needle 20 and is guided on an inner surface of the first pipe 32 in a slidable manner. A spring 35 is disposed between the armature core 31 and an adjust pipe 34 adjustably fixed on an inner surface of the stator core 30. The needle 20 has an annular contact portion 21 and a flat end surface 20a on its bottom end and is guided on an inner surface of the valve body 13. The annular contact portion 21 contacts with a valve seat 14a formed on an inner surface 14 of the valve body 13.

Referring to FIG. 1 and FIG. 2, the inner surface 14 provides a funnel-shaped fuel passage 50 of which a cross section decreases toward a downstream side. The inner surface 14 defines an opening 14b at a downstream end. A diameter of the opening 14b is smaller than that of the annular contact portion 21. The valve body 13 has a shallow and circular shaped depression 15 on its bottom surface. The depression 15 has a diameter 201 larger than that of the opening 14b. A cylindrical outer wall and a flat bottom surface 15a surrounding the opening 14b

define the depression 15.

A circular plate 25 is fixed on a bottom surface 13a of the valve body 13 by a laser welding. The plate 25 covers the depression 15 and defines a chamber 52 between the plate 25 and the valve body 13. The chamber 52 is thin, circular-shaped, and extended parallel with the plate 25. The plate 25 provides an approximately flat wall defining a downstream wall of the chamber 51. The plate 25 provides the flat wall extending throughout the chamber 51. The chamber 52 is divided into an inner chamber 52 and an outer chamber 53 by a projected line 200. The projected line 200 is defined by projecting the opening 14a on the plate 25 in an axial direction.

The plate 25 has a plurality of through holes 25a, 25b, 25c, and 25d as fuel orifices for defining a flow rate of fuel. The through holes 25a to 25d have the same diameter  $d_1$  and are arranged on a circle having a larger diameter than that of the contact portion 21 and the projected line 200. Each of the through holes is inclined to apart from an axis 26 of the plate 25 and the injector 1. The through holes 25a and 25b are inclined at the same angle  $\alpha$  and the through holes 25c and 25d are inclined at the same angle  $\alpha$  in an opposite direction. Therefore, the injector 1 provides two directional fuel injections. In this embodiment, the inclined angle  $\alpha$  is set within  $2^\circ$  to  $40^\circ$  ( $2^\circ \leq \alpha \leq 40^\circ$ ).

Each of the through holes 25a to 25d has an inlet opened between the projected line 200 and an outer line 201. Therefore, the inlets of the through holes 25a to 25d faces the bottom surface 15a of the valve body 13 and are shaded in an axial direction. Each of the through holes 25a to 25d has an outlet opened between the projected line 200 and the outer line 201. The inlet of each through holes 25a to 25d is disposed

apart more than the diameter  $d_1$  from the outer line 201. In this embodiment, a significant distance  $d_2$  ( $d_1 \leq d_2$ ) is provided in an inclining direction of the each through holes and in a radial direction. Therefore, the chamber 52 is extended more than the diameter  $d_1$  at an outside of the through holes.

When the coil 41 is not energized, the spring 35 pushes the needle 20 toward the seat 14a, the seat 14a and the contact portion 21 closes the fuel passage 50.

When the coil 41 is energized, the coil 41 generates an electromagnetic force between the stator core 30 and the armature core 31 and attracts the armature 31 and the needle 20 to lift up the needle 20. Therefore, the fuel passage 50 is opened to inject fuel.

Fuel flowing into the chamber 51 is divided into a first flow toward a center of the chamber 51 and a second flow toward radial outside of the chamber 52. The first flow meets and collides at a center of the plate 25 and turns into the radial outside. As a result, the first flow has a lot of turbulences. A part of the second flow and the turned first flow reaches to the inlets of the through holes after flowing along the plate 25. A remaining part of the second flow and the turned first flow passes between the inlets of the through holes and reaches to the outer end of the chamber 51. After that, the remaining part of the second flow changes its direction and reaches to the inlets of the through holes. Here, a distance  $d_2$  is wider than the diameter of the through holes to provide a passage on an outer side which is sufficient to provide a counter flow flowing radially from an outside to an inside. Therefore, fuel guided along the plate 25 flows into the inlets from all directions evenly. Fuel collides at just above the inlets and makes a lot of



turbulences in the column of the injected fuel. Therefore, each of the columns of the injected fuel from the through holes 25a to 25d are atomized finely. Additionally, the columns of the injected fuel don't collide each other, since four through holes are separately arranged.

5           FIGS. 4 and 5 show a second embodiment of the present invention. Hereinafter, the same or equivalent component as the above-mentioned embodiment is indicated by the same reference numerals and characterizing portions of each embodiment will be explained.

10           In this embodiment, a depression is formed on an upper surface of the plate 60 to provide the chamber 51. The through holes 60a to 60d are similar to the through holes 25a to 25d of the first embodiment.

15           FIG. 6 shows a third embodiment of the present invention. In this embodiment, a plate 70 and a plate 75 are fixed on the bottom surface 13a of the valve body 13. The plate 70 has a depression and through holes which are similar to the second embodiment. The plate 75 is disposed between the valve body 13 and the plate 70 for providing an opening 75a having the same diameter as the opening 14b. The plate 70 has the through holes 70a to 70d similar to the thorough holes 25a to 25d of the first embodiment. In this embodiment, fuel guided by the inner  
20 surface 14a reaches more inner side of the chamber 51, and changes a flow direction. Further, it is possible to form the chamber precisely.

25           FIG. 7 and 8 show a fourth embodiment of the present invention. In this embodiment, the plate has four through holes 80a, 80b, 80c and 80d. The through holes 80a and 80b are arranged inside of an imaginary line 202 on an upper surface of the plate 80 and form inner through holes. The through holes 80c and 80d are arranged outside of the imaginary line 202 and form outer through holes. Here, the imaginary

line 202 is defined as a circular line where a line extended along the inner surface 14 crosses the upper surface of the plate 80. The imaginary line 202 also indicates a portion where fuel flowing along the inner surface 14 directly collides with the plate 80. Therefore, the  
5 imaginary line 202 appears inside of the projected line 200. The through hole 80a of the inner holes and the through hole 80c of the outer holes are inclined toward a left side. The through hole 80b of the inner holes and the through hole 80d of the outer holes are inclined toward a right side.

10 In this embodiment, fuel flowing along the inner surface 14 is divided into a first flow toward the inner holes 80a and 80b and a second flow toward the outer holes 80c and 80d. Here, each of a paired through holes 80a and 80c mainly receives opposed flows. Therefore, fuel jet  
15 formed by the thorough hole 80a is influenced by the first flow so that the jet inclines inside from an axis 82 of the hole 80a. On the other hand, fuel jet formed by the thorough hole 80c is influenced by the second flow so that the jet inclines outside from an axis 82 of the hole 80c. As a result, a pair of jets injected from a pair of holes 80a and 80c are separated to avoid a collision of the fuel jets. In the through  
20 holes 80b and 80d, the same function is achieved.

FIG. 9 shows a fifth embodiment of the present invention. In this embodiment, a plate 95 has ten through holes 95a to 95 95j. The through holes 95a to 95d form inner through holes. The through holes 95e to 95j form outer through holes. The through holes 95a, 95b, 95e, 95f  
25 and 95g form a group of through holes directed in a left side. The through holes 95c, 95d, 95h, 95i and 95j form a group of through holes directed in a right side. In this embodiment, inner through holes and

outer through holes being member of one group are distanced at least L1. The outer through holes being member of one group are distanced at least L3 which is wider than the distance L1. Therefore, a collision of the jets injected from the outer through holes is avoided even the second  
5 flow is influenced on both of the adjacent outer through holes.

FIG. 10 shows a sixth embodiment of the present invention. In this embodiment, a plate 100 has twelfth through holes 100a to 100k and 100m. The through holes 100a to 100d form inner through holes. The through holes 100e to 100k and 100m form outer through holes. The  
10 through holes 100a, 100b, 100e, 100f, 100g and 100h form a group of through holes directed in a left side. The through holes 100c, 100d, 100i, 100j, 100k and 100m form a group of through holes directed in a right side. In this embodiment, the inner through holes being member of one group are distanced at least L2 which is wider than L1. Therefore, a  
15 collision of the jets injected from the inner through holes is avoided even the first flow is influenced on both of the adjacent inner through holes.

FIG. 11 shows a seventh embodiment of the present invention. In this embodiment, the needle is indicated by a reference 110. The contact  
20 portion is indicated by a reference 111. The needle 111 additionally has a protrusion 112 thereon. The protrusion 112 decreases a capacity of the inner chamber 52 and provides a flat wall facing the inlets of the inner through holes 80a and 80b. It is possible to reduce a remaining fuel in the chamber and improve an accuracy of a fuel measurement. Such a  
25 protrusion may be used for the above-mentioned embodiments.

Although the present invention has been described in connection with the preferred embodiments thereof with reference to the accompanying

drawings, it is to be noted that various changes and modifications will be apparent to those skilled in the art. Such changes and modifications are to be understood as being included within the scope of the present invention as defined in the appended claims.